16: Mechanical and Electrical Vibrations (section 3.7)

Consider linear dynamical system in which mathematical model is the following IVP:

$$au'' + bu' + cu = g(t), \quad u(0) = u_0, \quad u'(0) = v_0.$$

Here g(t) is **forcing function** of the system. Here we only discuss the case g(t) = 0, i.e free vibrations, no external force. A solution u(t) of the ODE on an interval containing t = 0 that satisfies the initial conditions is called the **response** of the system.

Spring/mass systems: Free Undamped Vibration (or simple harmonic motion)

1. Spring on a table (horizontal spring).)

$$mu'' = -ku$$

- 2. A flexible spring is suspended vertically from a rigid support and the mass m is attached to the end. By **Hooke's Law**, the spring itself exerts a restoring force F opposite to the direction of elongation and proportional to the amount of elongation L: F = -kL, where k is called the **spring constant**.
- 3. The mass m stretches the spring by L and attains a position of equilibrium, i.e. weight, mg, is balanced by the restoring force:

$$mg - kL = 0.$$

4. If the mass is displaced by an amount u from its equilibrium position, the restoring force is then -k(u+L). Free motion (i.e. no other external/retarding forces acting on the moving mass): use Newton's second Law with the net (or resultant) force:

$$mu'' = -k(u+L) + mq = -ku.$$

5. ODE of Free Undamped Motion:

$$u'' + \omega_0^2 u = 0, \tag{1}$$

where

$$\omega_0^2 = \frac{k}{m}.$$

Initial conditions: $u(0) = u_0$, $u'(0) = v_0$, where u_0 is the initial displacement and v_0 is the initial velocity. For example, $u_0 < 0$ and $v_0 = 0$ mean that the mass is released from rest from a point $|u_0|$ units above the equilibrium position.

General solution of (1) is

$$u(t) = C_1 \cos(\omega_0 t) + C_2 \sin(\omega_0 t) = R \cos(\omega_0 t - \delta),$$

where

• $R = \sqrt{C_1^2 + C_2^2}$ is called the **amplitude** of the motion

• δ is called the **phase**, or phase angle, and measures the displacement of the wave from its normal position corresponding to $\delta = 0$. Recall that

$$\cos \delta = \frac{C_1}{\sqrt{C_1^2 + C_2^2}} = \frac{C_1}{R}, \quad \sin \delta = \frac{C_2}{\sqrt{C_1^2 + C_2^2}} = \frac{C_2}{R}.$$

- $\omega_0 = \sqrt{\frac{k}{m}}$ is the **natural frequency** of the system.
- $T = \frac{2\pi}{\omega_0}$ is the **period** of the motion. The number T is time it takes the mass to execute one cycle of motion (the length of the interval between two successive maxima (or minima) of u(t).)
- The frequency of motion $f = \frac{1}{T} = \frac{\omega_0}{2\pi}$.
- 6. A mass weighing 4lb stretches a spring 6 inches. At t=0 the mass released from a point 8 inches below the equilibrium with an upward velocity of $\frac{2}{3}$ ft/s. Determine the amplitude of vibrations, phase angle, period, natural frequency of the system and frequency of motion.

Spring/mass systems: Free Damped Vibrations.

- 7. Assume that the mass is suspended in a viscous medium or connected to a dashpot damping device. Dampers work to counteract any movement: damping force $= -\gamma v = -\gamma u'$, where γ is a positive damping constant.
- 8. ODE of Free Damped Motion:

$$mu'' + \gamma u' + ku = 0. (2)$$

9. Discriminant of the characteristic equation $m\lambda^2 + \gamma\lambda + k = 0$ is

$$D = \gamma^2 - 4mk.$$

10. CASE 1: (Underdamping) D < 0, i.e. the roots are complex conjugate:

$$\lambda_{1,2} = -\frac{\gamma}{2m} \pm i\sqrt{\omega_0^2 - \frac{\gamma^2}{4m^2}},$$

where $\omega_0^2 = \frac{k}{m}$.

General solution of (2) is not periodic:

$$u(t) = e^{-\frac{\gamma}{2m}t} \left(\cos(\omega t) + C_2 \sin(\omega t) \right) = Re^{-\frac{\gamma}{2m}t} \cos(\omega t - \delta),$$

where

• $Re^{-\frac{\gamma}{2m}t}$ is damped amplitude of vibrations.

- $\omega = \sqrt{\omega_0^2 \frac{\gamma^2}{4m^2}}$ is the **quasi frequency**; here $\omega_0^2 = \frac{k}{m}$, ω_0 is the **natural frequency** of the system (i.e. without damping). Note that $\omega < \omega_0$, if there is a damping.
- $T_d = \frac{2\pi}{\omega}$ is the **quasi period**, i.e. the time interval between two successive maxima of u(t).

Note that as the damping coefficient γ increases, the quasi frequency μ becomes smaller and the quasi period becomes bigger.

11. CASE 2: (Critical Damping) D = 0 (two repeated (equal) roots) or equivalently

$$\gamma_{\rm crit} = 2\sqrt{mk}$$
.

The motivation for the term "critical":

- (a) If $0 < \gamma < \gamma_{\text{crit}}$, then D > 0 the motion is **oscillatory** in the following sense: the mass passes the equilibrium infinitely many times, although the amplitude of oscillation decays exponentially;
- (b) (Overdamping) If $\gamma > \gamma_{\rm crit}$, then D > 0 (two distinct real roots λ_1 and λ_2). In this case there are no oscillation. The general solution of (2) has no more than one zero:

$$x(t) = C_1 e^{\lambda_1 t} + C_2 e^{\lambda_2 t}.$$

so the motion is **non-oscillatory**.

Conclusion: The critical dumping is the transitional value of the damping from oscillatory to non-oscillatory motion.

For $\gamma = \gamma_{\text{crit}}$ the characteristic equation has repeated roots. The general solution of equation of type (2) will be discussed in the next section.

LRC electrical circuit

12. If Q is the charge at time t in an electrical closed circuit with inductance L, resistance R, and capacitance C, then by Kirchhoff's Second Law (from Physics) the impressed voltage E(t) is equal to the sum of the voltage drops in the rest of the circuit

$$E(t) = IR + \frac{Q}{C} + LI'(t).$$

By substitution I = Q' we get

$$LQ'' + RQ' + \frac{1}{C}Q = E(t).$$

Analogy between electrical and mechanical quantities:

Resistance R Damping constant γ Inverse capacitance 1/C Spring constant kImpressed voltage E(t) (electromotive force) External force F(t)